



**focus on energy<sup>sm</sup>**

Partnering with Wisconsin utilities

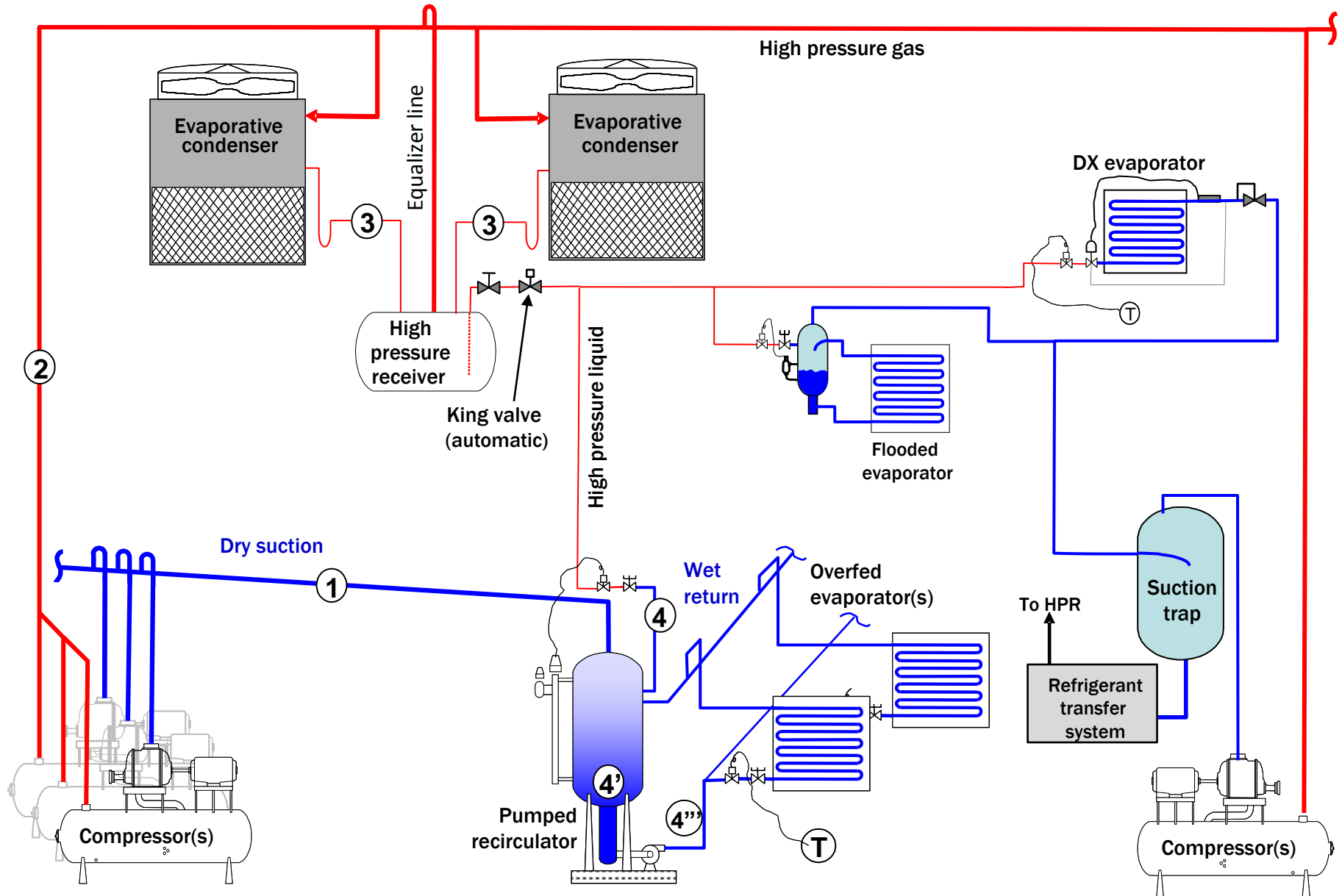
# Applying Variable Speed Drives In Industrial Refrigeration Systems

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Consortium

# Single Stage Compression System



# Where Can VFDs Be Applied?

- **Condenser fans**
  - All or none – **expect 2-3% savings**
- **Evaporator fans**
  - **2-4% savings range**
  - Simple paybacks 1-5 years
- **Compressors**
  - At most, one VFD comp per suction level
  - **Simple paybacks 1-4 years**

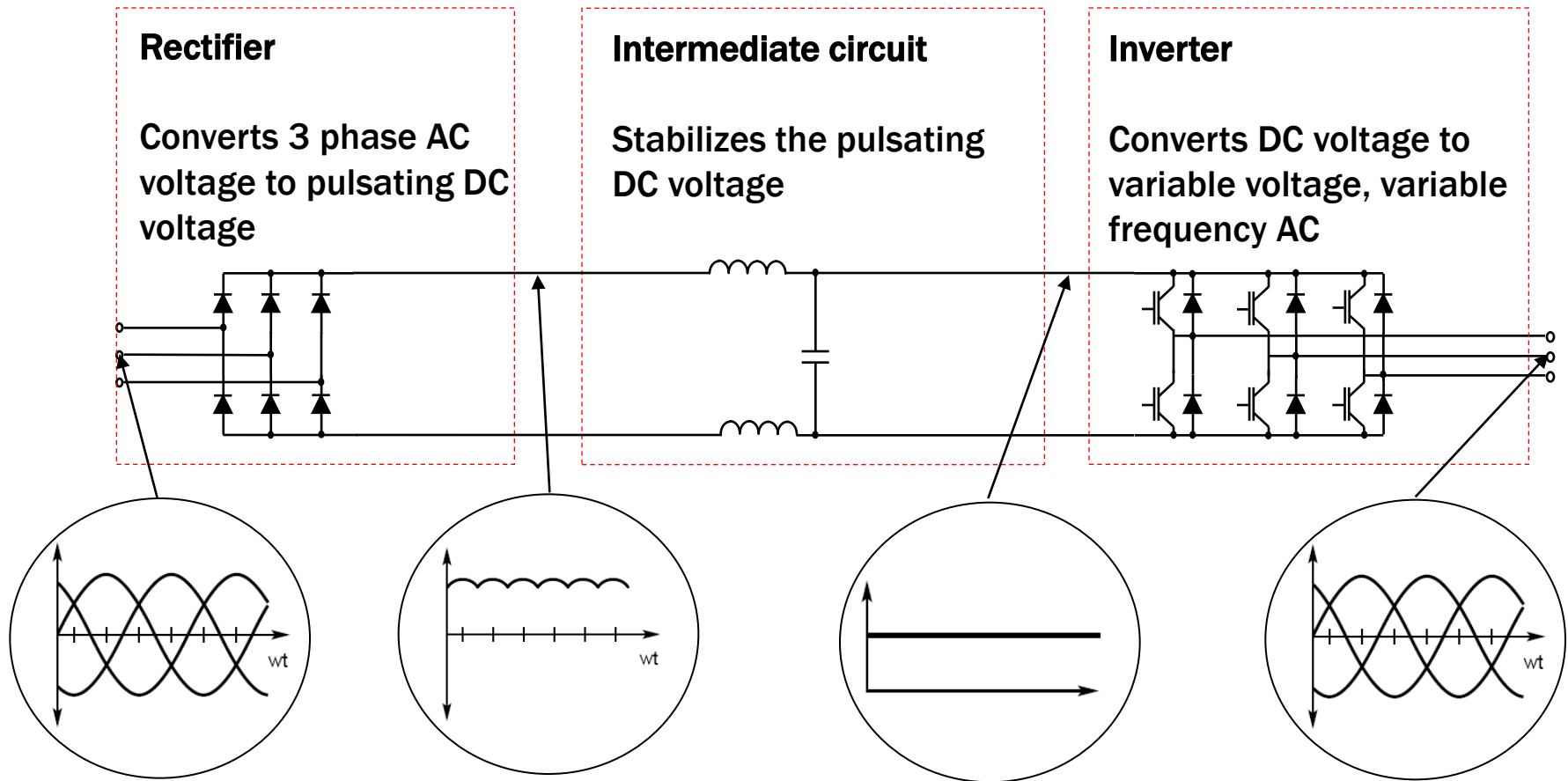


# VFD Basic Principles

- Common terms – AFD, ASD, VFD, VSD, drive
- What it is – an electronic controller device capable of adjusting speed of electric motor by modulating power being delivered
- What it can do:
  - Vary motor speed to match variable load requirement
  - Soft start to eliminate mechanical, electrical, hydraulic surges/transients
  - Potentially save energy, energy cost



# How VFDs Work



# VFD Basic Principles

- **Rating**
  - Voltage
  - Required current to motor
  - Load type
  - Overload requirements
- **Variable torque** – required torque increases as speed increases (condenser, evaporator fans)
- **Constant torque** – required torque independent of speed (compressors)

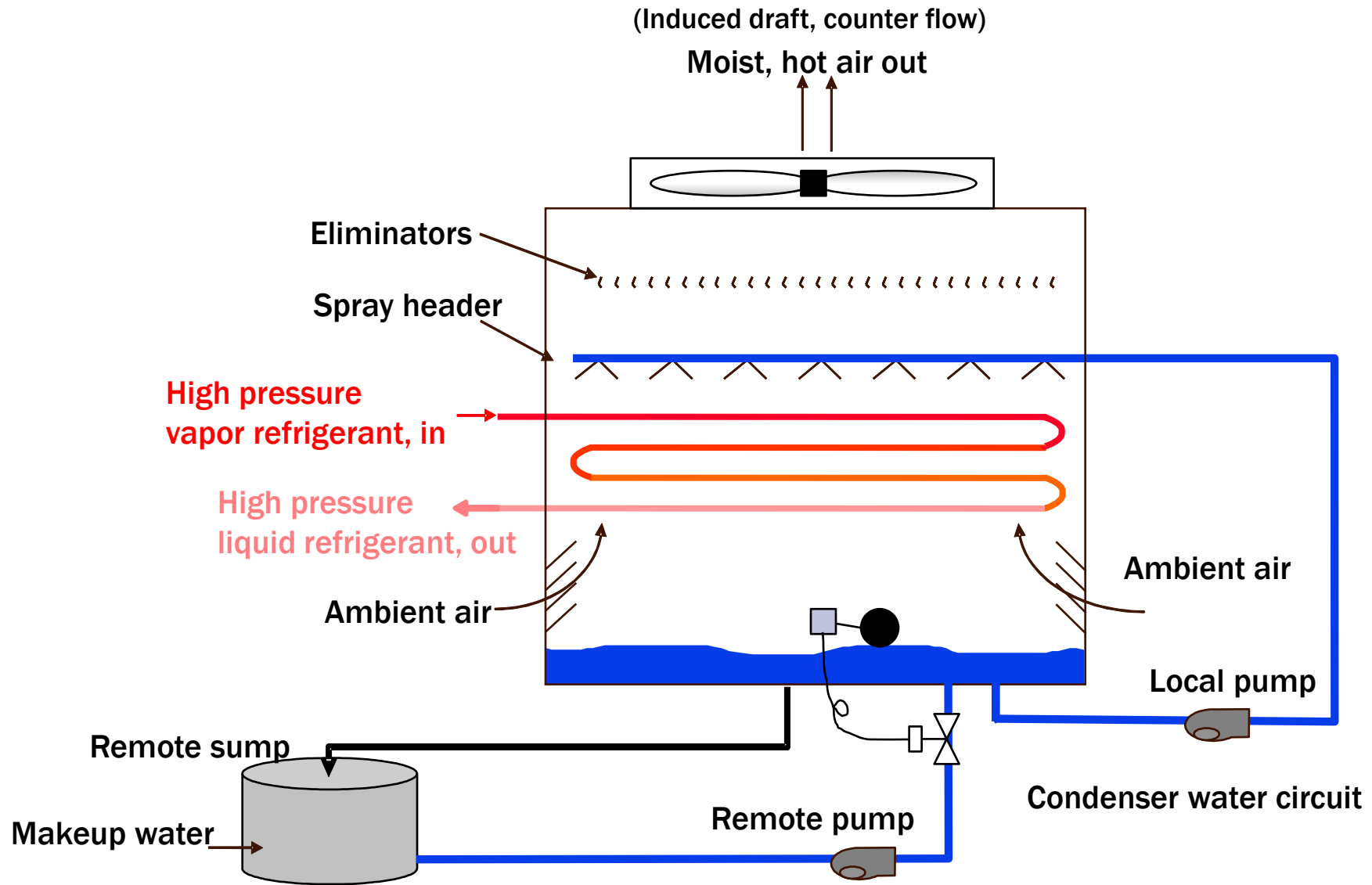


# VFDs – Condenser Fans



**A variable torque application.**

# Evaporative Condenser Operating Principles





# Condenser Fan Control Options

- **Single speed** fan with on/off control
  - Most common method of head pressure control
  - Need to set cut-in (e.g. 150 psig), cut-out pressures (e.g. 145 psig)
  - Simple control method except:
    - ☐ Highest energy consumption option
    - ☐ Higher maintenance (fan motors, belts)
    - ☐ Potential for liquid management problems in multiple condenser systems
- **Two-speed** fan control

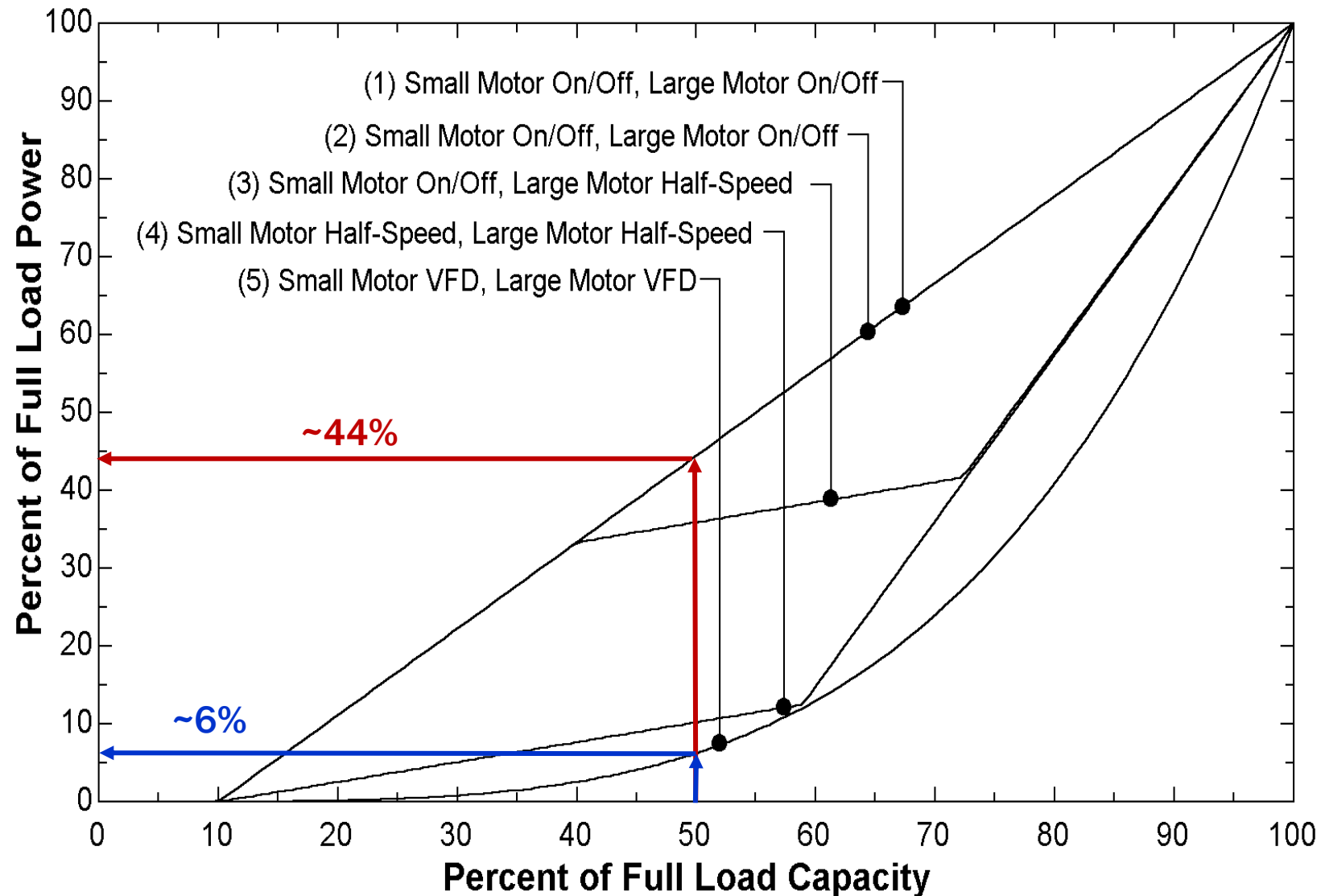
# Condenser Fan Control Options (continued)

- **Variable frequency drive**
  - Set target head pressure, modulate fan speed to maintain
  - Simple to implement:
    - Slightly higher capital cost versus fixed speed
    - Lowest energy consumption control alternative
    - Multiple condenser systems, modulate ALL condensers together
    - Smooth system operation with minimal transients

# Condenser Fan Control Map Example

Strategy		Mode 1	Mode 2	Mode 3	Mode 4	Mode 5
1	Small Motor	off	on	off	on	
	Large Motor	off	off	on	on	
2	Small Motor	off	off	on		
	Large Motor	off	on	on		
3	Small Motor	off	on	on	on	
	Large Motor	off	off	half-speed	on	
4	Small Motor	off	half-speed	half-speed	on	on
	Large Motor	off	off	half-speed	half-speed	on
5	Small Motor	off	variable speed			
	Large Motor	off	variable speed			

# Comparative Condenser Fan Performance



# Simple Two Condenser System

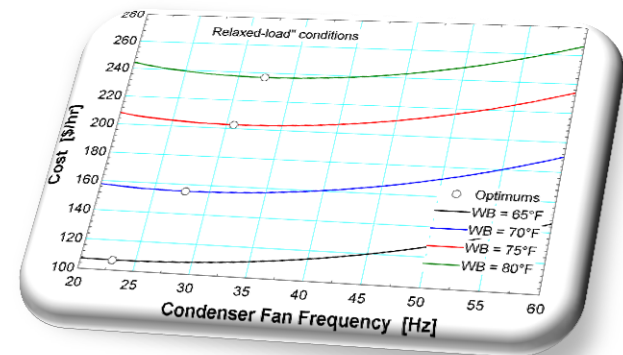
Heat rejection load	Fixed speed control		Variable speed drive	
	# condensers	HP	# condensers	HP*
100%	2	30	2 @ 100%	30
75%	1 + 1/2	21.6	2 @ 75%	9.8
50%	1	15	2 @ 50%	1.8

\* Sans drive losses

Each condenser equipped with 15 HP fan.

# VFDs – Condenser Fans

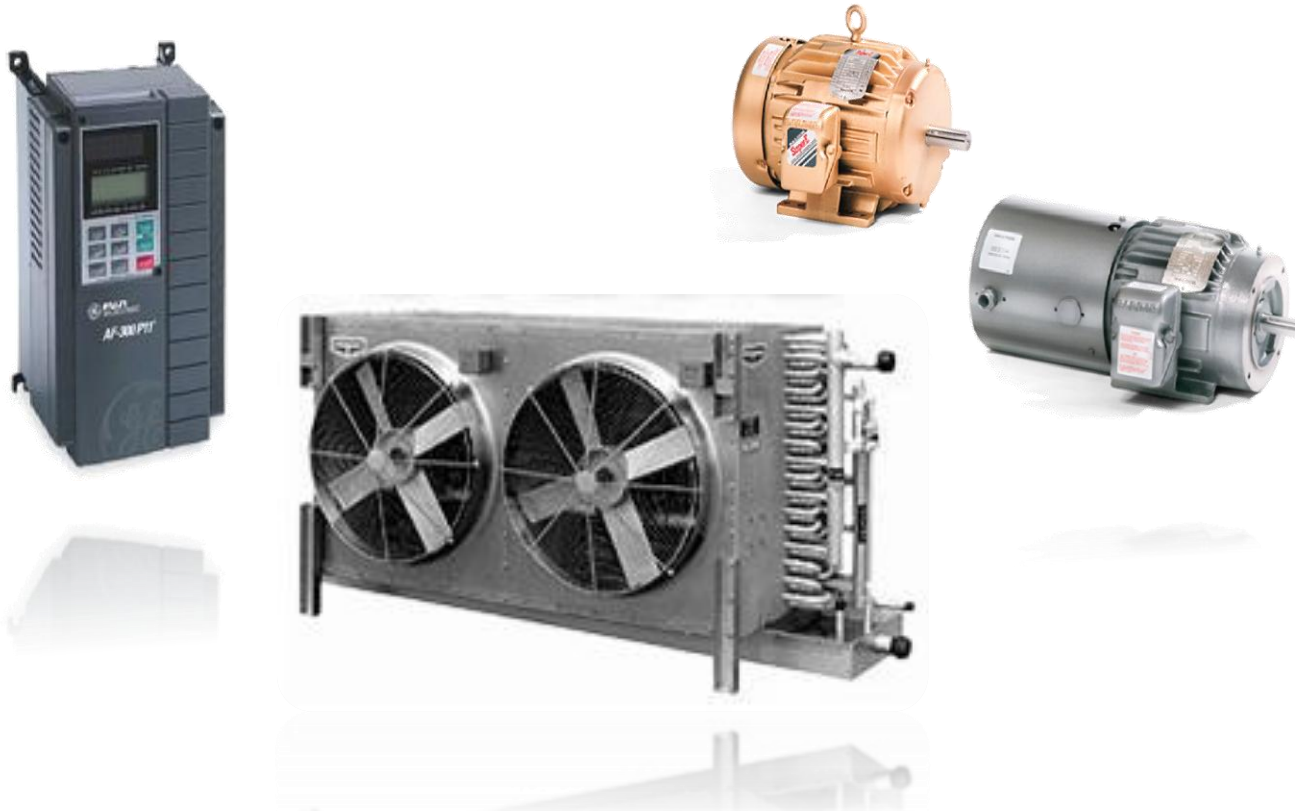
- **Benefits**
  - Stable head pressure
  - Reduced maintenance on shafts, bearings, belts
  - Allows ability to optimize head pressure
  - Reduced energy consumption and operating costs
- **Look to achieve optimal/near-optimal performance with a simplified control strategy**



# Questions on VFDs for condenser fans?



# VFDs – Evaporator Fans



**A variable torque application.**

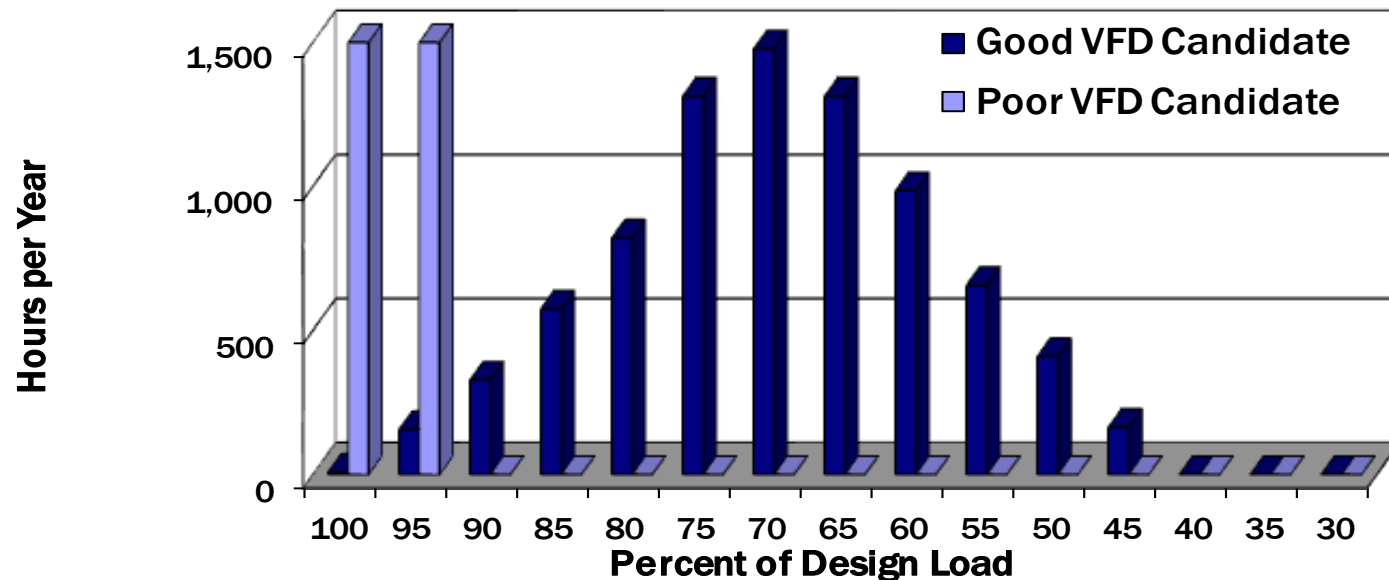


# Evaporator Fans Opportunity

- Load variability essential
- Best choices for modulating evaporator capacity as space load is reduced:
  - Cycle refrigerant feed, always run fans (normal situation)
  - Cycle refrigerant feed + cycle fans after period of time with no call for refrigerant feed (duty cycle)
  - Raise suction pressure, always run fans (often not possible)
  - Cycle refrigerant feed with VFD control frequency (best practice)
- Which is best?

# Good Applications For Evaporator Fan VFDs

- Larger fan motors
- High hours/year operation
- Frequent part-load operation



# First Principles

- Applicable fan laws

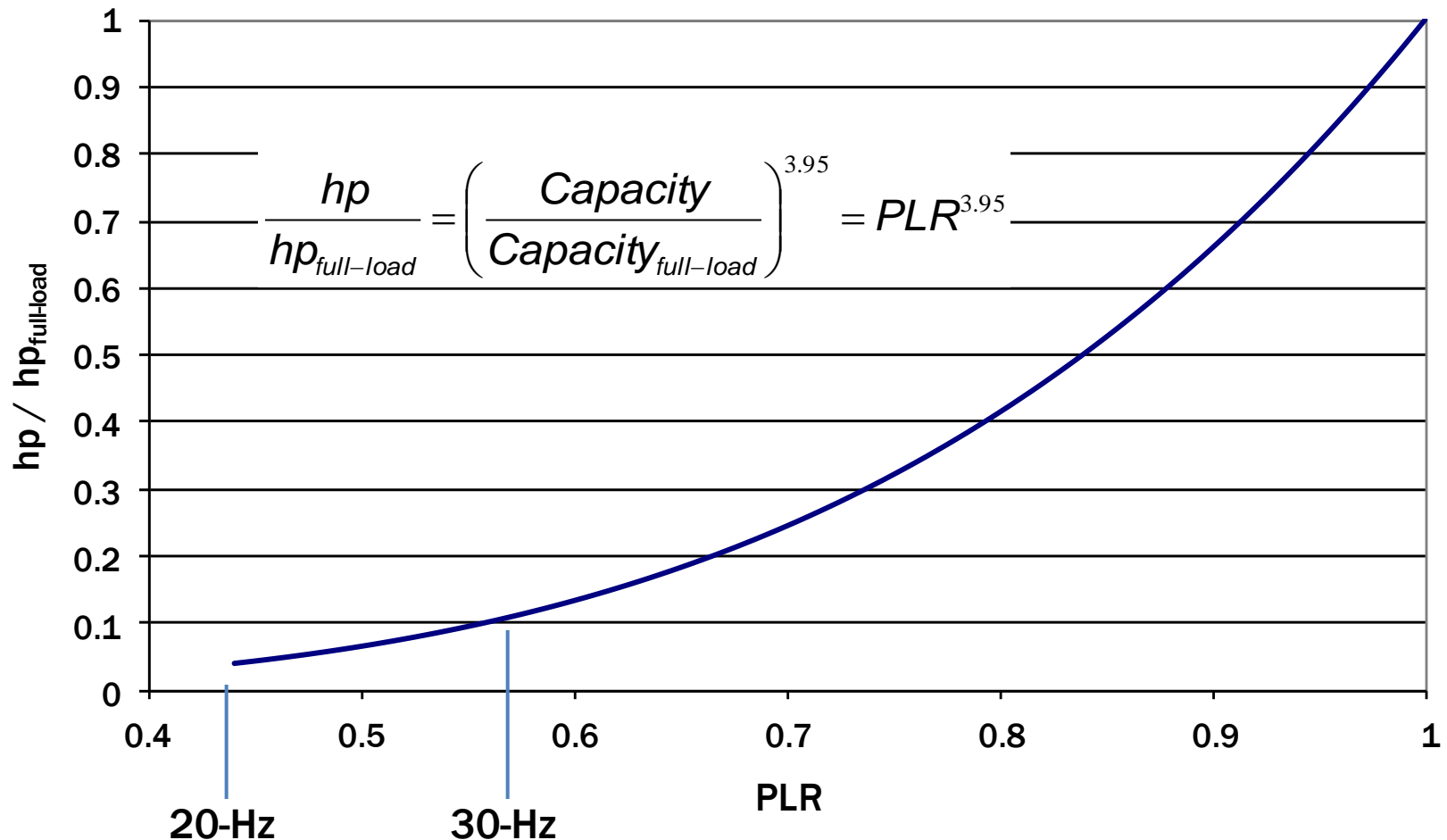
$$\frac{N}{N_{full-load}} = \frac{CFM}{CFM_{full-load}}$$

$$\frac{hp}{hp_{full-load}} = \left( \frac{CFM}{CFM_{full-load}} \right)^3$$

- Limitations – typical minimum motor speeds between 20-30 Hz
- Heat exchange impact

$$\frac{Capacity}{Capacity_{full-load}} = \left( \frac{CFM}{CFM_{full-load}} \right)^{0.76}$$

# Speed Control Impact On Fan Power

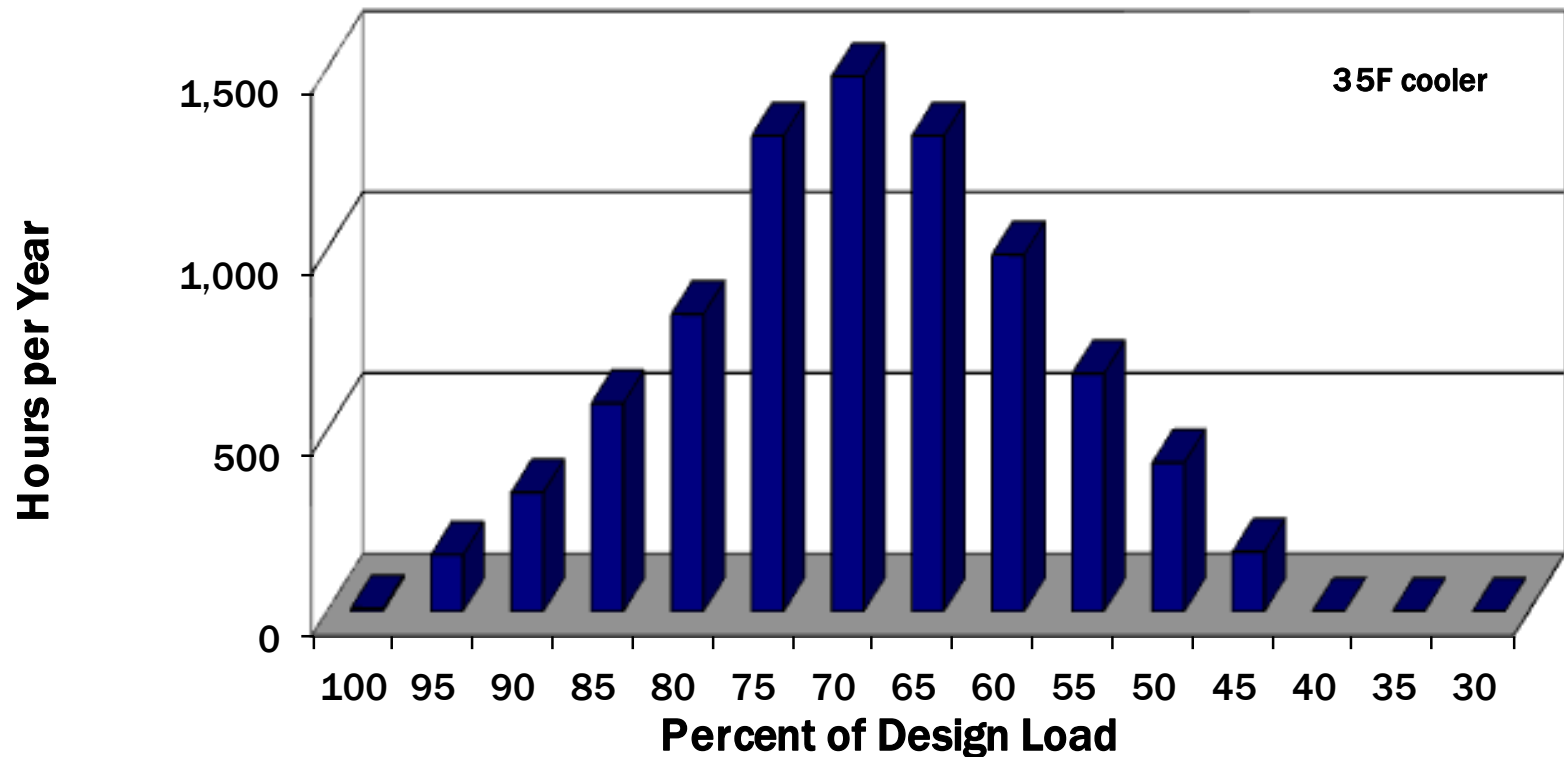


# Evaporator Fan VFD Savings Example

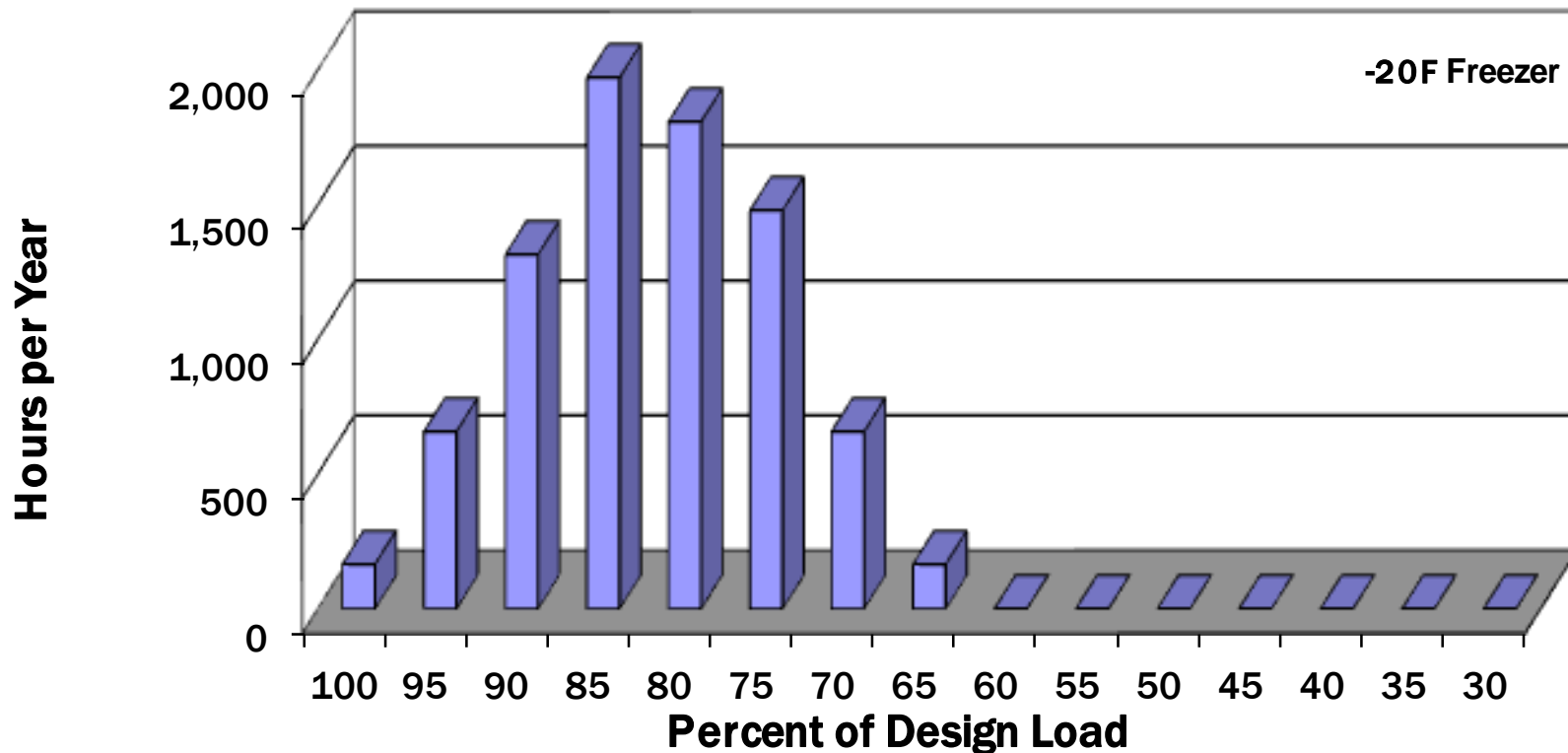
- **Evaporator – TD\* = 12°F for cooler and 8°F for freezer**
- **VFD costs**
  - Assume 3 HP VFD for each evaporator
  - Installation 15 hours/VFD by electrician @ \$65/hour
- **Energy costs**
  - Blended \$0.08/kWh

\* TD = Temperature difference ( $T_{\text{room}} - T_{\text{refrigerant}}$ )

# Comparative Load Profiles – Cooler



# Comparative Load Profiles – Freezer

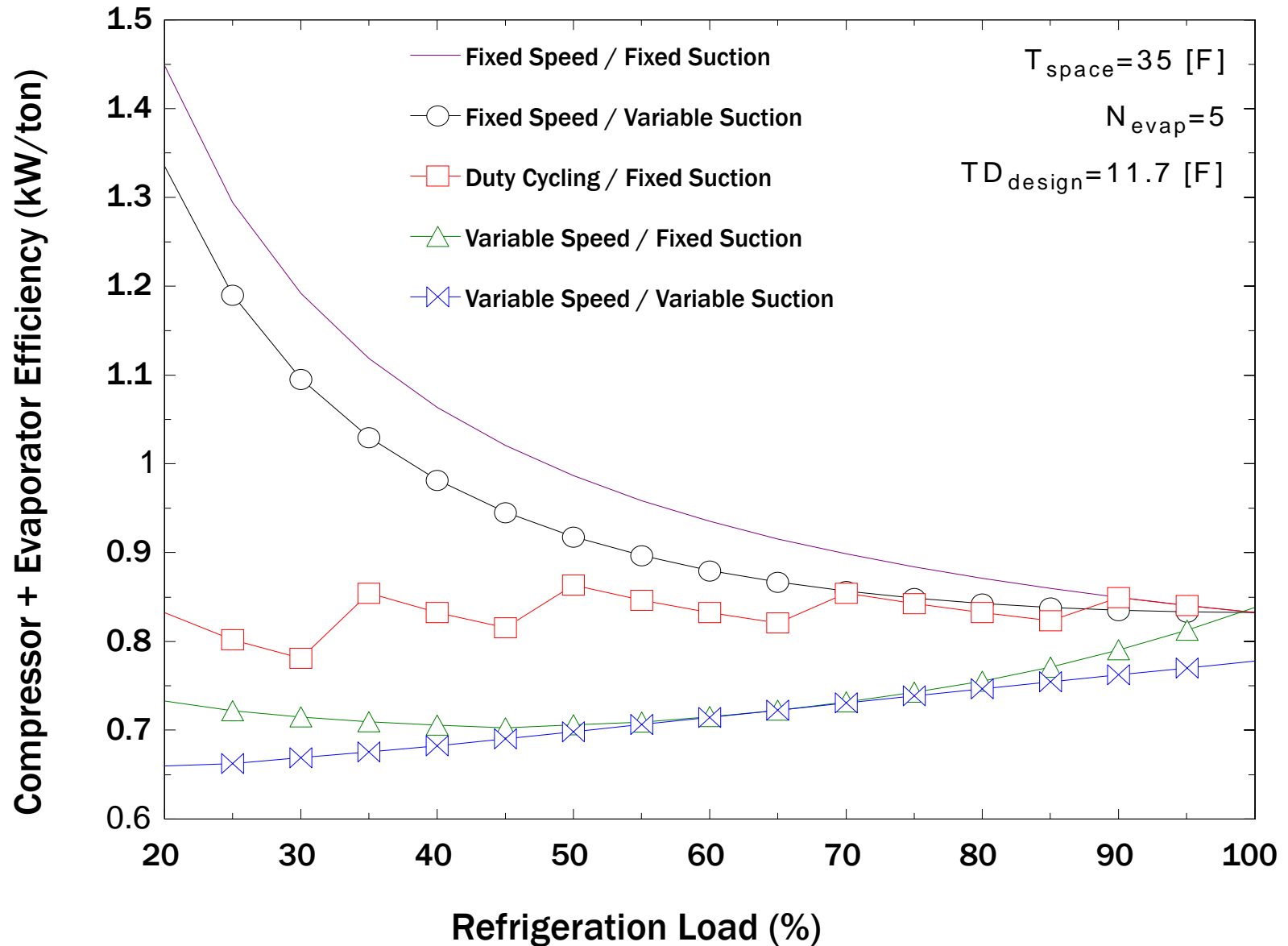


# Control Options

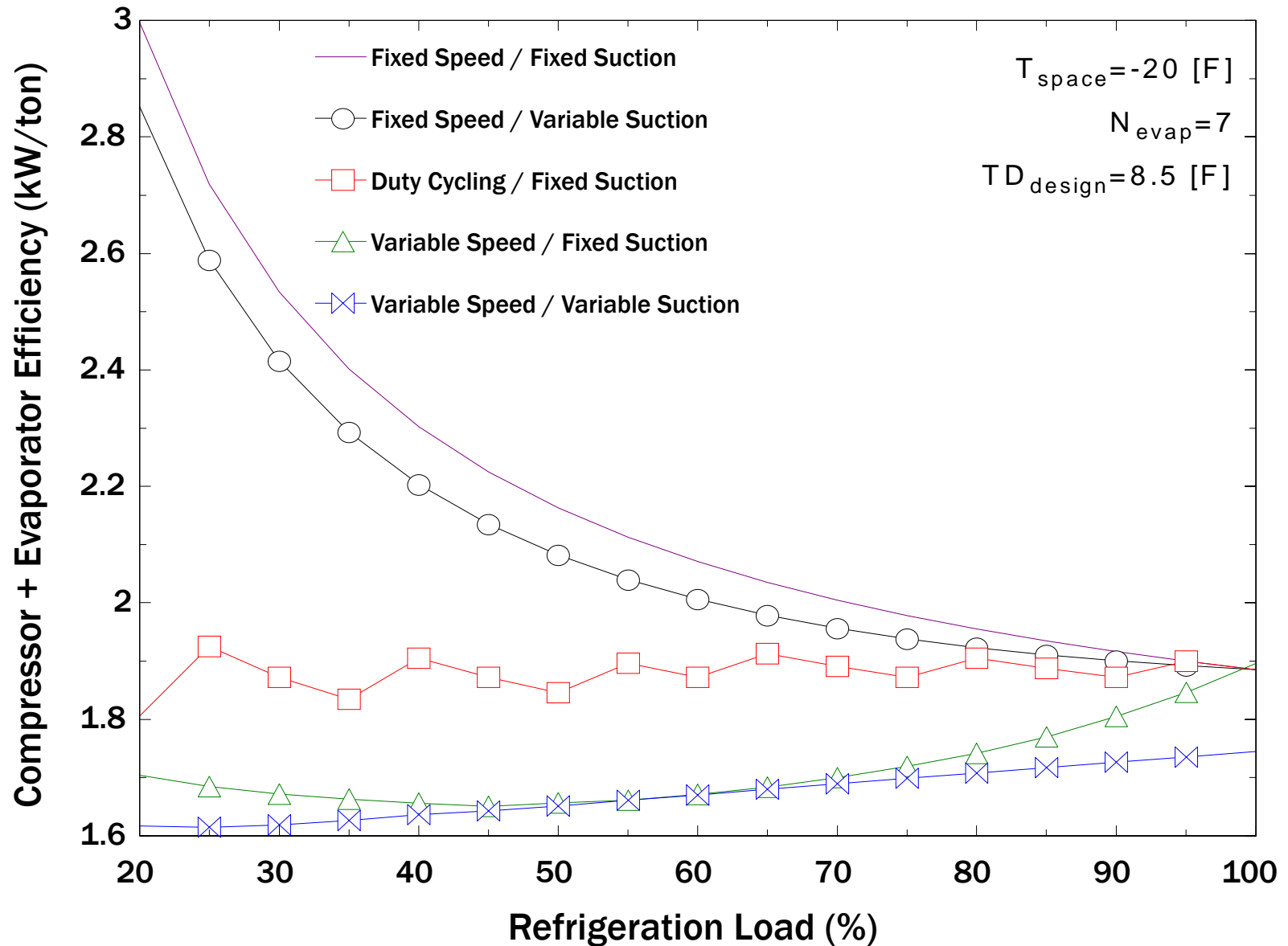
	Fan Speed Control	Suction Pressure Control
<b>#1</b>	<b>Fixed</b>	<b>Fixed</b>
<b>#2</b>	<b>Fixed</b>	<b>Variable</b>
<b>#3</b>	<b>Duty Cycle</b>	<b>Fixed</b>
<b>#4</b>	<b>Variable</b>	<b>Fixed</b>
<b>#5</b>	<b>Variable</b>	<b>Variable</b>



# Comparative Energy Performance – Cooler



# Comparative Energy Performance - Freezer



# Economic Analysis Results

	Cooler (35°F)	Freezer (-20°F)
<b>From always on fan control to VFD</b>		
Savings per ton	\$72	\$96
Capital cost per ton <sup>†</sup>	\$65	\$95
Installation cost per ton	\$48	\$71
<b>Simple payback</b>	<b>1.6 years</b>	<b>1.7 years</b>
<b>From cycling fan control to VFD</b>		
Savings per ton	\$48	\$64
<b>Simple payback</b>	<b>2.4 years</b>	<b>2.6 years</b>

<sup>†</sup> Assumes single drive operates all fan motors (4) on individual evaporators

# VFDs – Evaporator Fans

- Reduced system power
  - Evaporator fan power drastically reduced at part-load
  - Compressor power reduced due to lower parasitic space refrigeration **loads** attributable to lower fan energy use (5 HP = 1 ton of refrigeration)
- More stable suction pressure
- Increased motor life
  - Less motor cycling, inherently soft-start
- Improved power factor
- Decreased noise, “wind-chill”

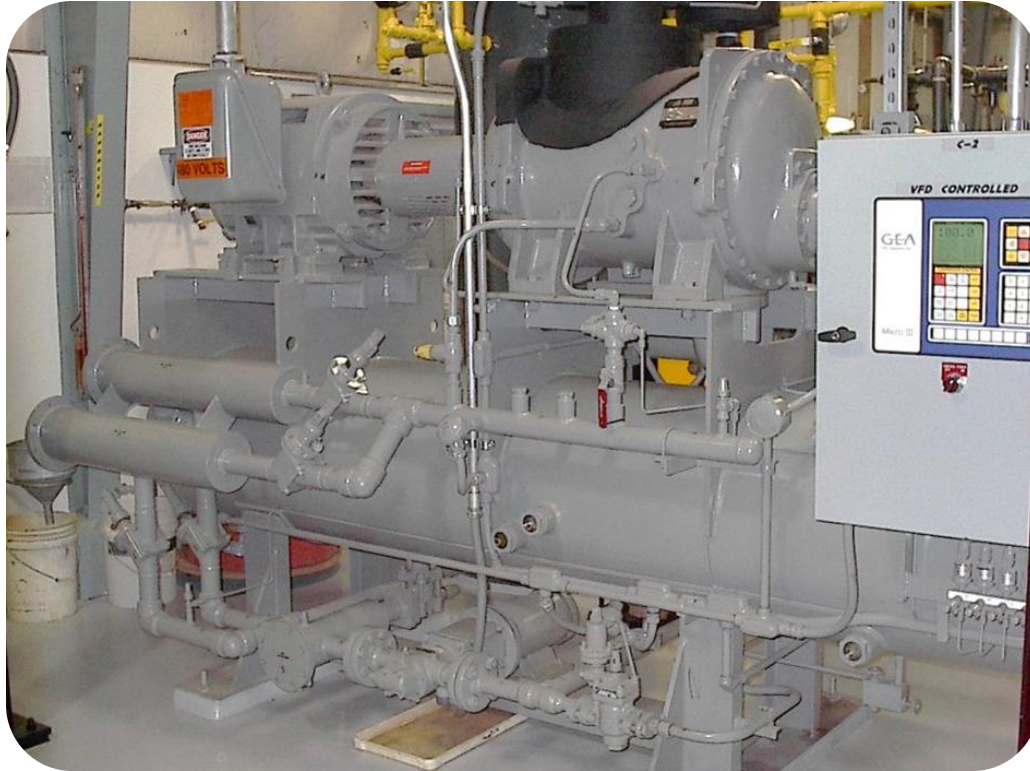
# VFD 'Heads Up'

- Drive losses (~3-6%, losses increase at low loads)
- Evaporator “throw” loss
- Typical systems have large number of small evaporator fan motors (cost)
- Additional equipment to maintain
- Resonance of equipment (natural frequency)
- Power quality
- Siting the drives
- Minimum fan speed/drive frequency
- Requires low temperature bearing lubrication/grease

# Questions on VFDs for evaporator fans?



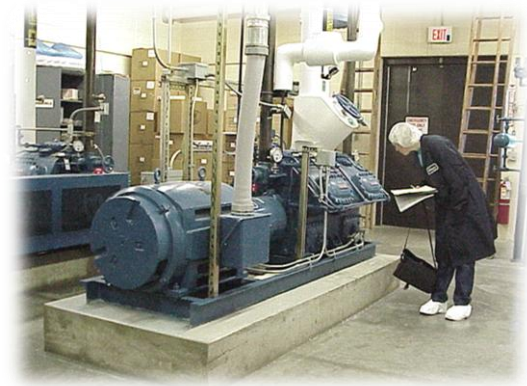
# VFDs – Compressors



**A fixed torque application.**

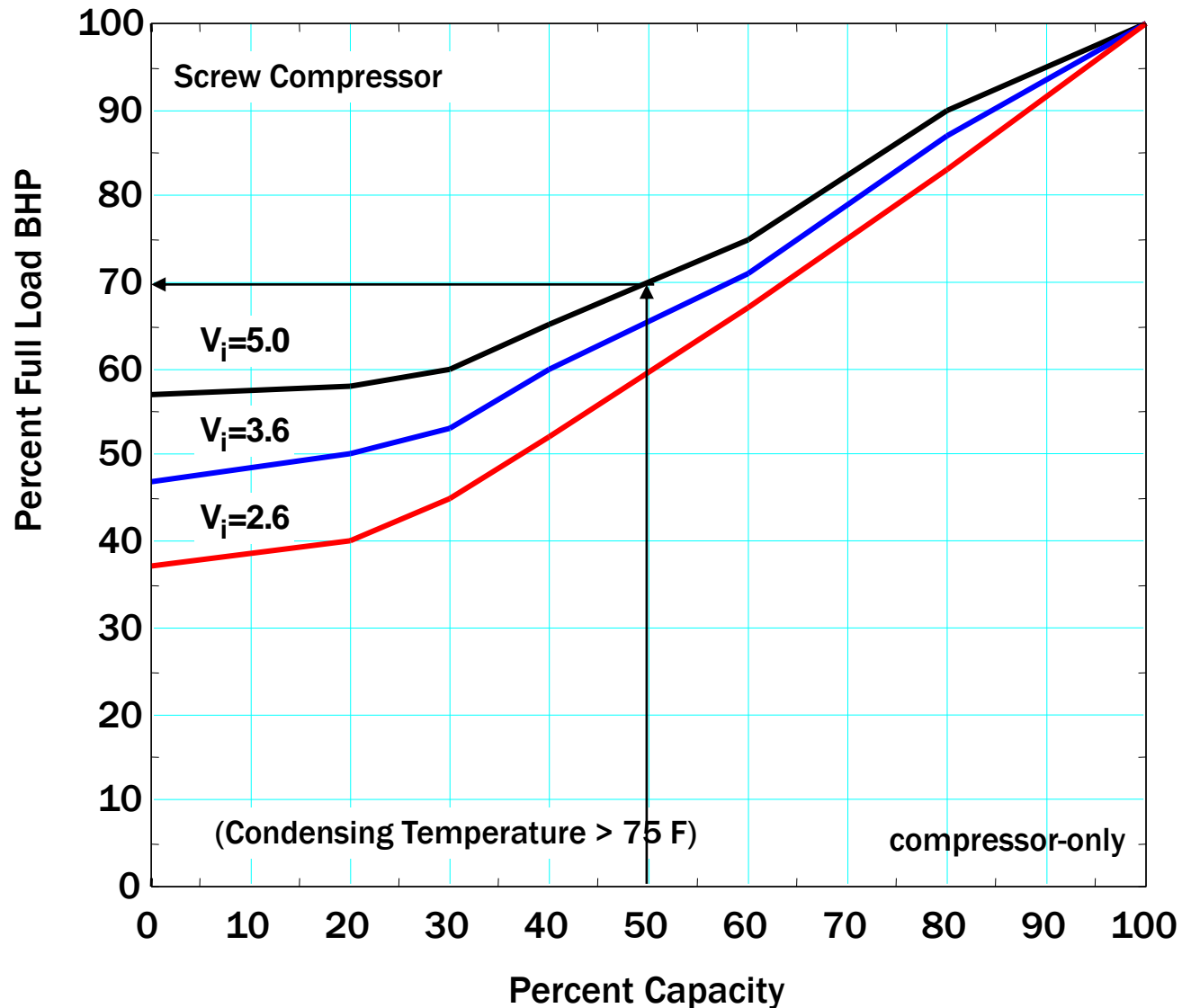
# Compressor Capacity Control

- **Reciprocating**
  - Start/stop individual compressors (rack system)
  - Discrete cylinder unloaders
  - Hot gas bypass (not preferred)
  - Variable speed drive?
- **Screw – single, twin**
  - Continuous slide valve, poppet valves, etc.
  - Hot gas bypass (not preferred)
  - Variable speed drive?

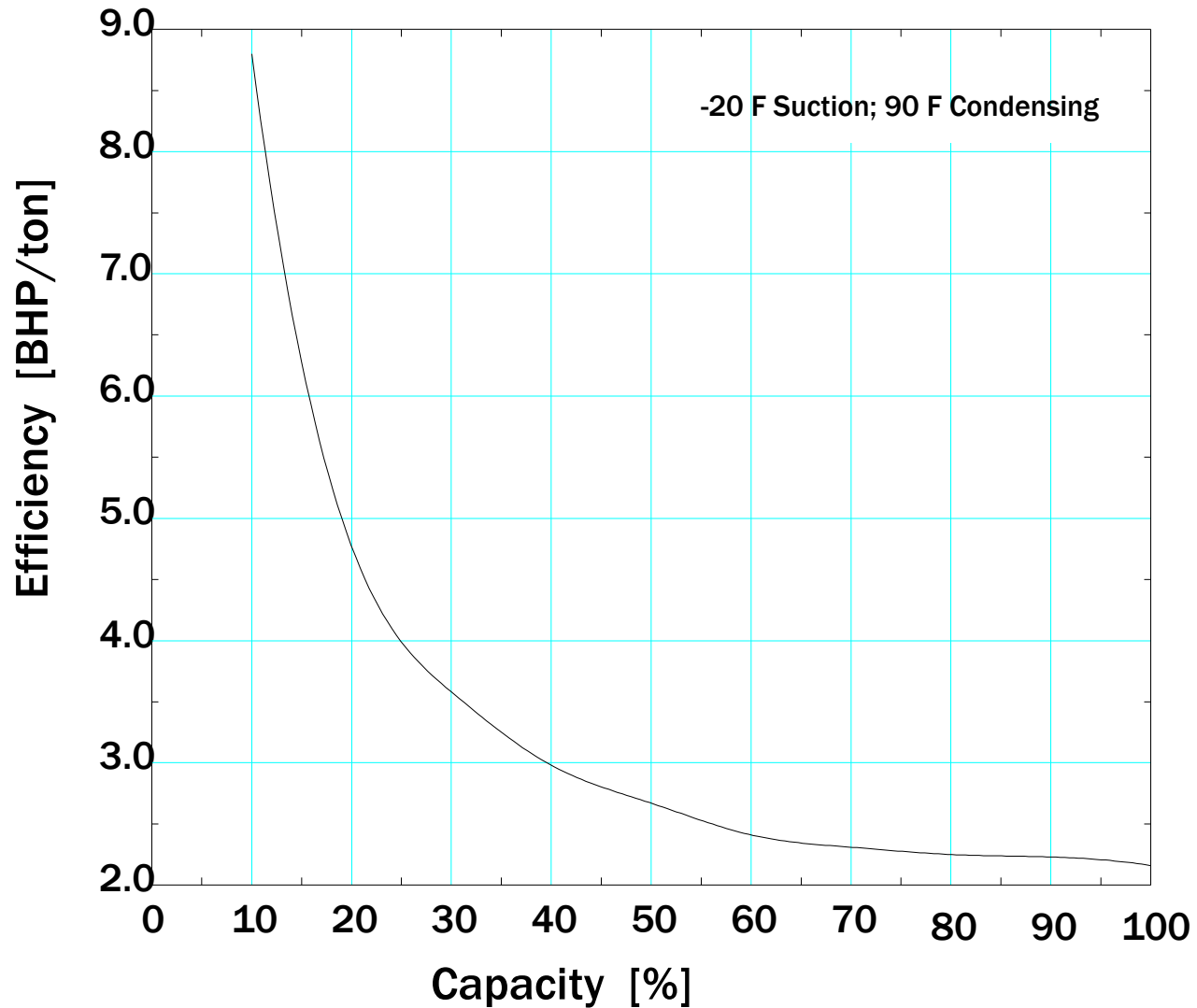




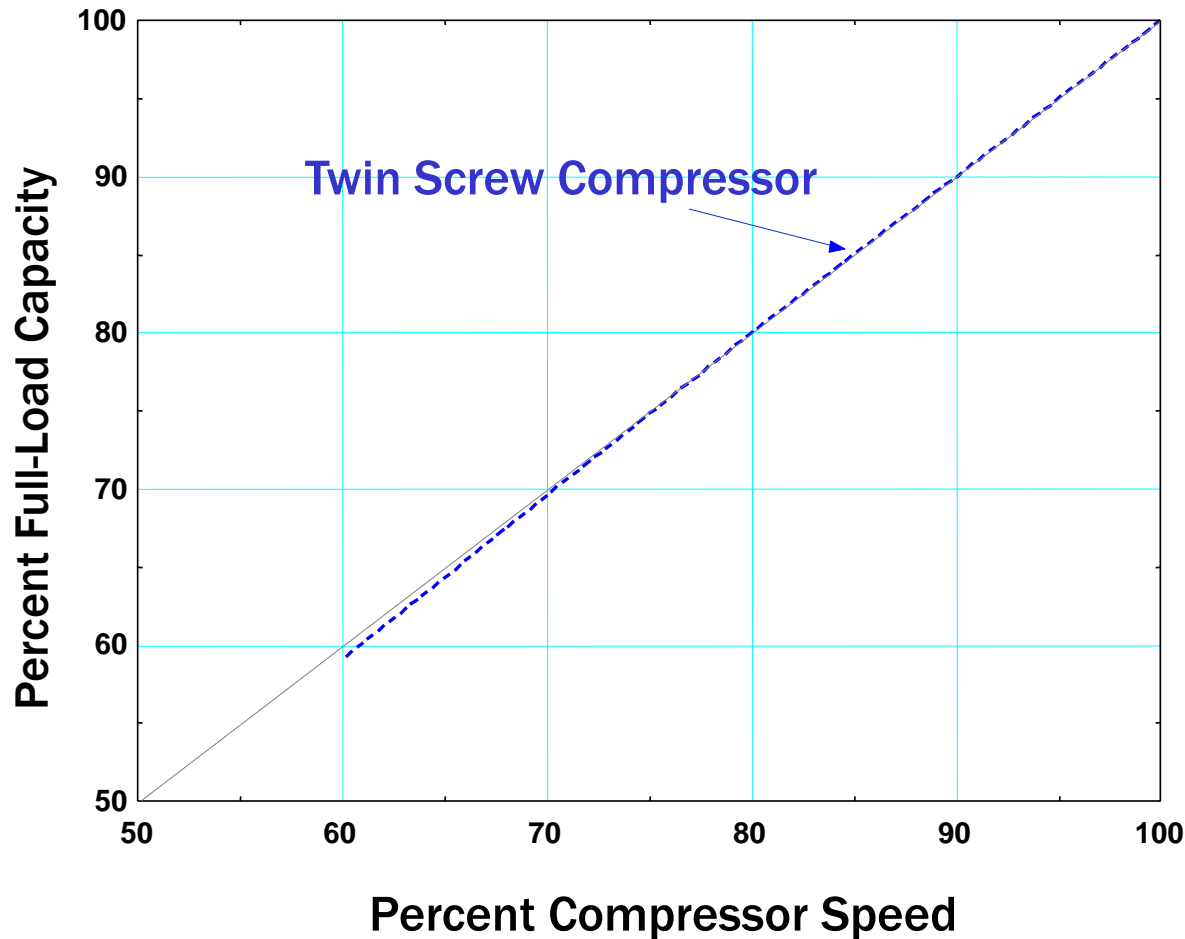
# Screw Compressor Part-load Characteristics



# Inefficient Part-load Operation!



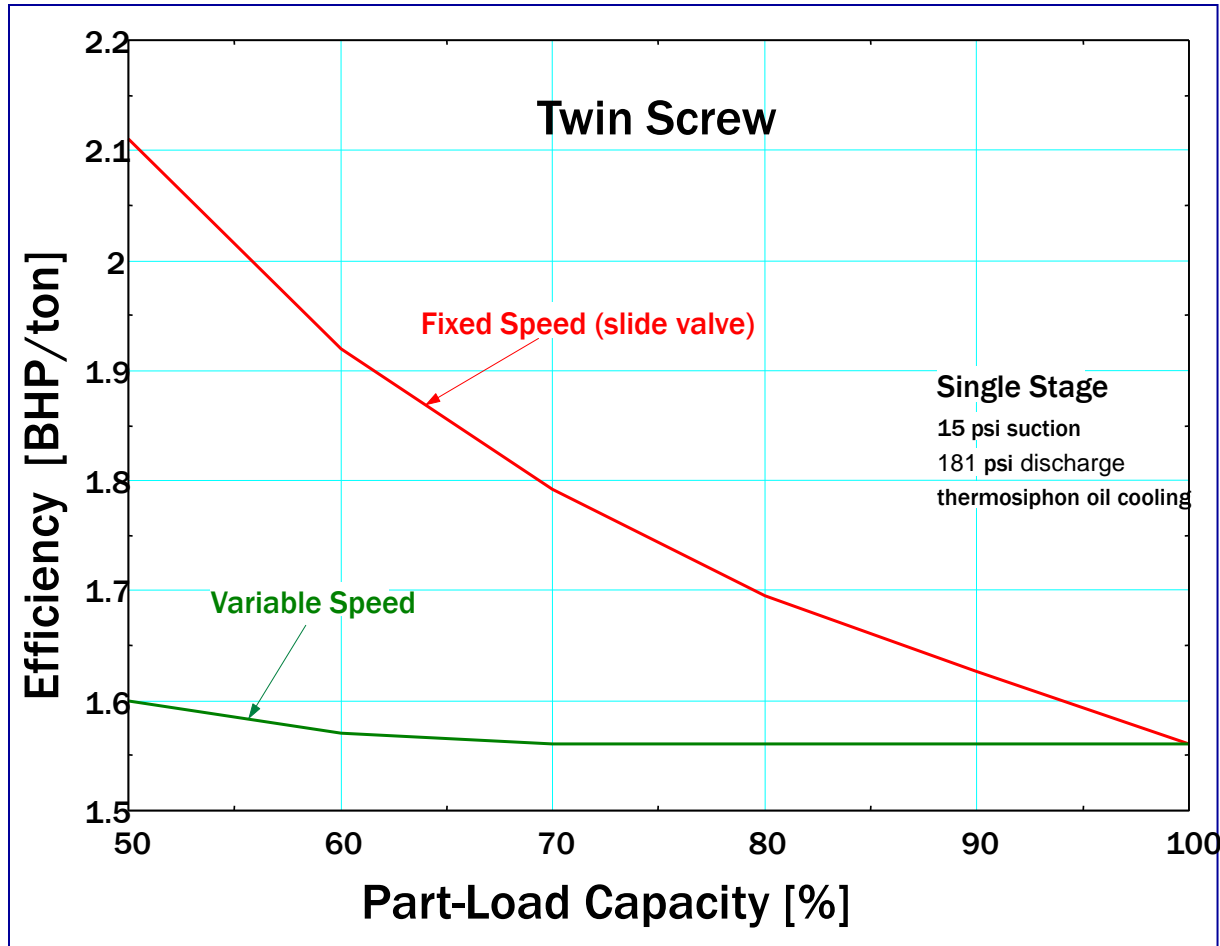
# Compressor Capacity Control



**Compressor capacity is directly proportional to shaft speed**



# Speed Control Efficiency Benefit

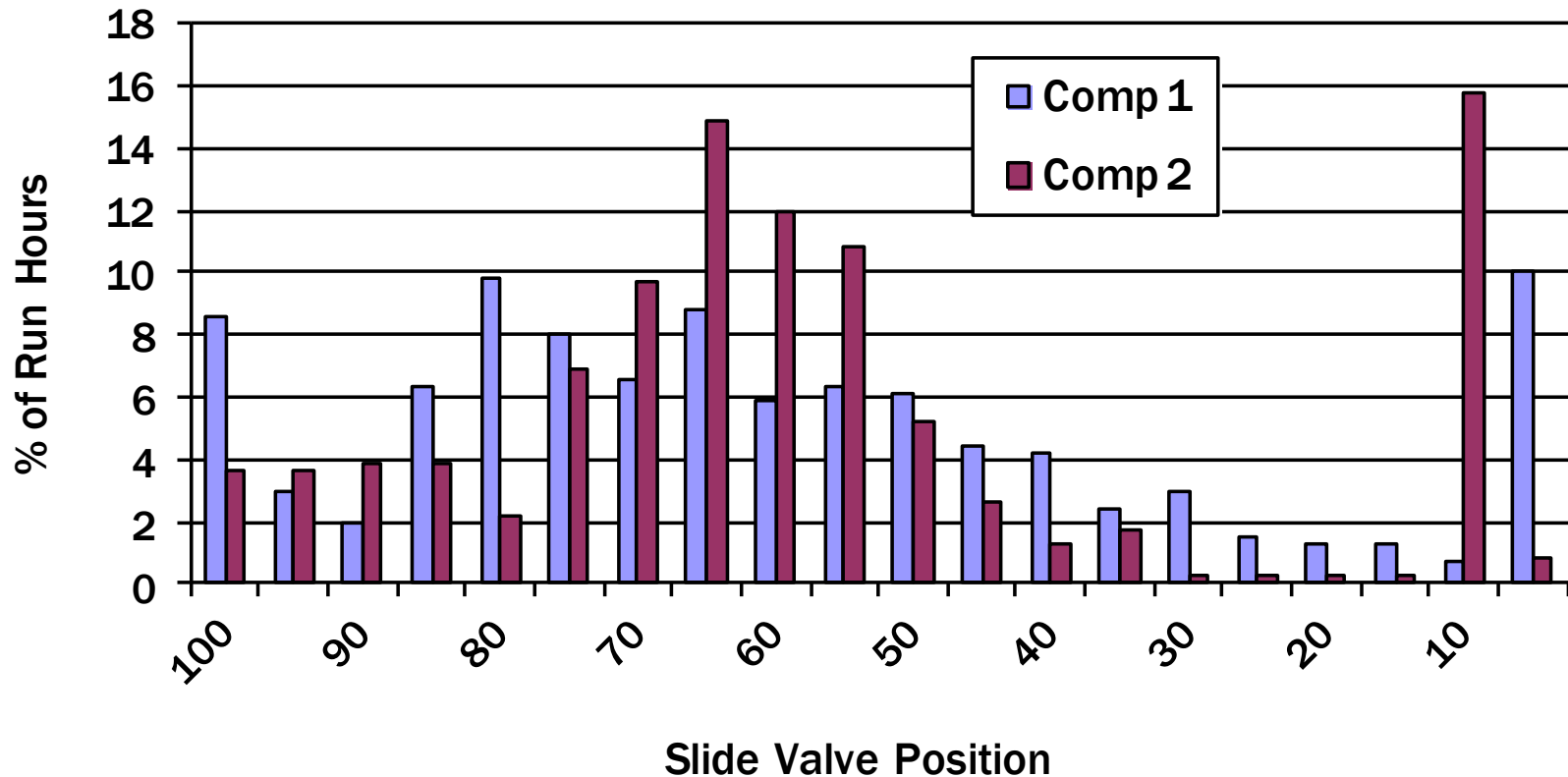


**VFDs perform  
well at part-  
load  
conditions!**



# Screw Compressor Field Retrofit

Compressors operation on -30°F suction level



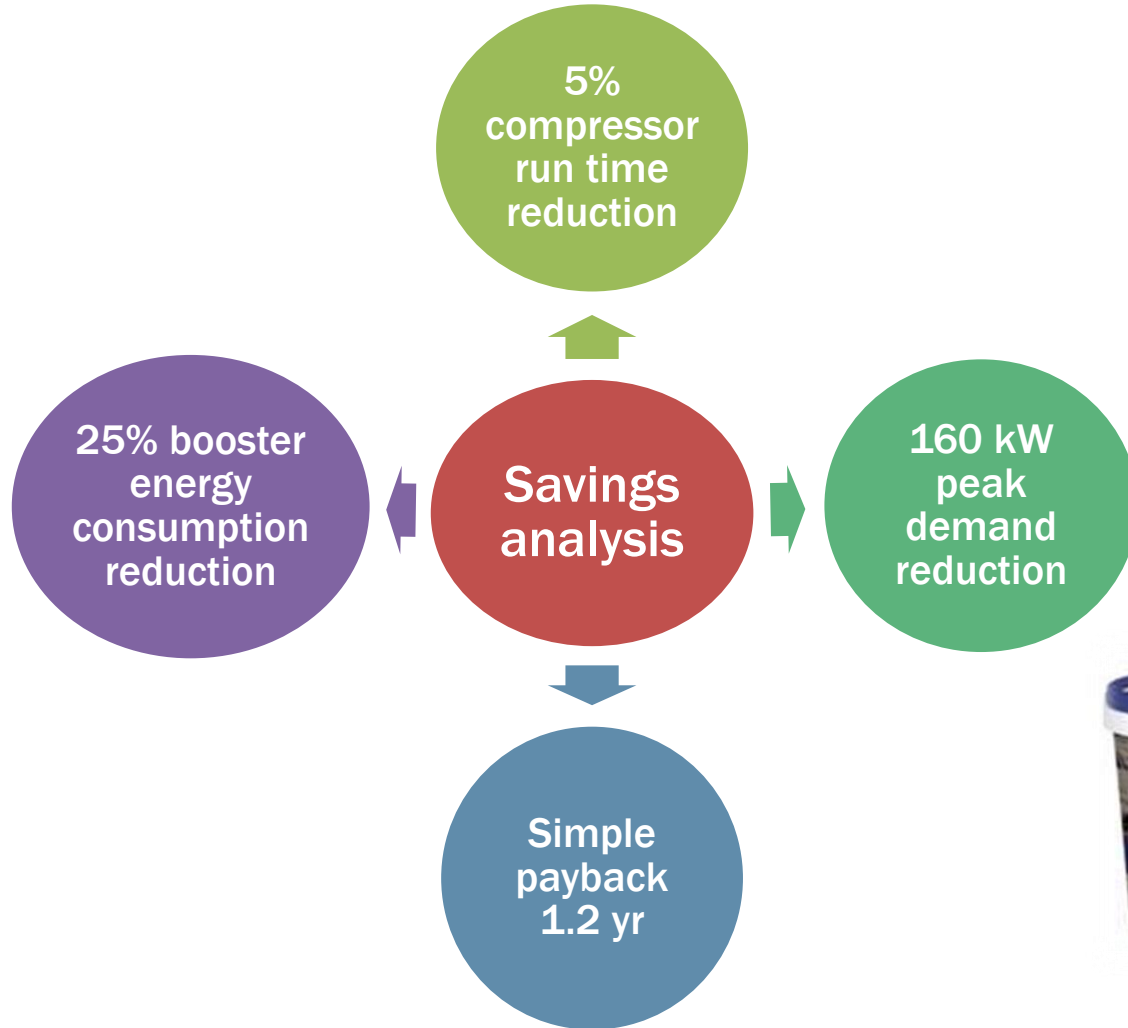
# Compressor VFD Retrofit

- Installed 500 HP VFD on -30 suction level (trim machine)
- Implemented 1<sup>st</sup> level of capacity control using speed rather than slide valve
- All other compressors base loaded at 100%



500 HP inverter rated motor (North Ice Cream Plant)

# Compressor VFD Retrofit Positive Results



# Compressor VFD Application Considerations

- One VFD-equipped compressor per suction level
- Operating sequence considerations
  - Base load fixed speed screws at 100% slide valve
  - Trim with VFD-equipped compressor
    - ☐ Use speed as first level of capacity control
    - ☐ Use slide valve as second level of capacity control
  - Monitor, tune PI control to avoid speed cycling



# Compressor VFD Application Considerations

(continued)

- Not cost effective for medium voltage (4160 V) motors
- Speed control range – essential to field verify compressor is free from excessive vibration at all frequencies/speeds
- Adhere to minimum speed
  - Required for adequate oil circulation
  - Required to maintain rotor tip velocity
  - May require supplemental fan for motor cooling



# Questions on VFDs for compressors?



# Closing Thoughts

- **Condenser fans**
  - All or none – expect 2-3% savings
- **Evaporator fans**
  - 2-4% savings range
  - Simple paybacks range 1-5 years
- **Compressors**
  - At most, one VFD comp per suction level
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# Questions?

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**Thank you!**



# Additional Resources

Industrial Refrigeration Consortium – [www.irc.wisc.edu](http://www.irc.wisc.edu)

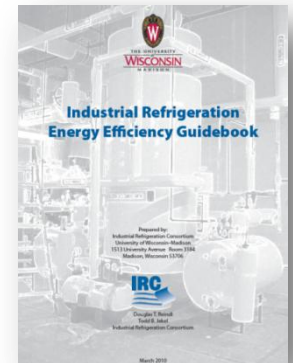
## Research and Technology Forum Presentations

- “Fundamentals of VFDs and Refrigeration Applications” Ohme (2009)
- “VFDs for Evaporators” Runsey (2009)
- “VFDs for Compressors” Cosner (2009)
- “Case Studies of VFD Applications” Zanutto (2009)
- “VFDs in Industrial Refrigeration – Lesson’s Learned” Campbell (2006)

## Cold Front Newsletters

- “VFDs for Evaporative Condenser Fans” Vol. 4 No. 2 (2004)
- “VFDs for Screw Compressors” Vol. 4 No. 3 (2004)
- “VFDs for Evaporator Fans” Vol. 4 No. 4 (2004)

Industrial Refrigeration Energy Efficiency Guidebook,  
Reindl & Jekel, 2<sup>nd</sup> Edition, (2011)



# Additional Resources (continued)

## University of Wisconsin-Madison/IRC

- Short course – “Achieving Energy Cost Savings for Ammonia Refrigeration Systems” May 22-24, 2012
- <http://epd.engr.wisc.edu/ammoniarefrigeration>

## Northwest Energy Efficiency Alliance *Evaporator Fan VFD Initiative*

- Baseline Market Evaluation Report, April 1999
- Market Progress Evaluation Report No 2., November 2000
- Market Progress Evaluation Report No 2., June 2002
- Reports available at [www.nwalliance.org](http://www.nwalliance.org)

